# ROTOR BLADES MODELLING FOR A CENTRIFUGAL COMPRESSOR USING ANALYTICAL METHODS 

Nicuşor BAROIU, Virgil Gabriel TEODOR, Florin SUSAC, Nicolae OANCEA<br>Department of Manufacturing Engineering, "Dunărea de Jos" University of Galaţi, florin.susac@ugal.ro


#### Abstract

In this paper we propose an analytical method for rotor blade modelling for a centrifugal compressor. The considered rotor allows as hub surface a revolution surface with circular axial generatrix. The machining is made with a pre-formed tool with ball end. In this case, the contact with blade surface takes place both on the cylindrical and the spherical zone of the mill. A graphical solution developed in CATIA is presented, for a helix with constant axial pitch.


Keywords: centrifugal compressor, CATIA modelling, pre-formed mill

## 1. INTRODUCTION

Generating blade surfaces of a rotor belongs to a centrifugal compressor and it ? is included in the frame of free form surfaces machining [1] and may be regarded based on the fundamental theorems of the surfaces generating by wrapping [2].

The complexity of geometrical form for the centrifugal compressor as well as the diversity of applications for this compressor types in aerospace constructions, [3], [4], require CNC machine tools with five numerical controlled axis for machining.

This imposes the tool path planning at the machining of these surfaces types. This is a main concern in machining.

Many studies and proposals were made in order to increase the machining yield.

Also, it is noted that the general NC programs are ineffective for machining centrifugal pumps [5] and it is proposed a mathematical model for tool path using an interactive algorithm.

In order to improve the machining technology, Wu et al. propose an innovative approach which puts together the machining and the numerical simulation of the generation of centrifugal pumps blades [6]. In the same paper, it is proposed a rapid method for designing this type of blades. The method is based on the cubic spline curves and use machine tools with five numerical controlled axes.

There were studied and proposed methods to avoid collision between cutting tool and blank, determining the tool trajectory based on the blade and hub geometric model.

In this paper, we propose a method for blade generating, developed in analytic form, based on the generating trajectories, for machining a rotor of centrifugal compressor. It was developed a specific algorithm for a hub with circular generatrix. Also, it was developed, in CATIA, a graphical model of generating with end ball mill.

## 2. GENRATING KINEMATIX. REFERENCES SYTEMS

In what follows, it is studied the analytic modelling of blade generating for a hub with revolution body, with radius $R_{0}$, see figure 1 .

In figure 1 are presented the reference systems associated with the rotor support and the cutting tool (end mill tool with composed profile: straight line and circular profile at tool end). The tool axis is inclined regarding the rotation axis and the tool end has radius $r_{0}$.

It is defined the radius of the axial hub profile, $R_{0}$. The circle's arc is defined between the normals tangent to the $\overparen{M N}$ arc, see figure 2 . The points $M$ and $N$ represent the contact points of the $\overparen{M N}$ arc with the hub limit's, with radii $R_{b}$ and $R_{t}$.


Fig. 1. Hub with axial circular profile; reference systems


Fig. 2. Axial hub's section

The normals to the $\overparen{M N}$ arc, in $M$ and $N$ limits determine the centre of the circle's arc, point $O$.

The values of $R_{0}, R_{b}$ and $R_{t}$ radii are accepted as constructive values.

Are defined the following reference systems:

- $X Y Z$ global reference system, with $Z$ axis joined with hub;
$-X_{0} Y_{0} Z_{0}$ mobile reference system, with origin in $O_{l}$, a current point onto the hub's circular generatrix;
- $X_{0}^{\prime} Y_{0}^{\prime} Z_{0}^{\prime}$ mobile reference system, initially overlapped to $X Y Z$, joined with $X_{0} Y_{0} Z_{0}$ in the helical motion with $\vec{V}$ axis and $p$ helical parameter. The $\vec{V}$ axis is overlapped to $Z$ axis.
- $X_{I} Y_{l} Z_{l}$ reference system, with $X_{l}$ axis overlapped to peripheral surface of end mill tool.

The generating kinematics includes the rotation around the $\vec{V}(Z)$ axis, the movement $I$, linked with the translation movement $I I$, along the $\vec{V}$ axis.

The rotation of the end mill tool around the $\vec{A}$ axis, III movement, is a cutting motion. The tool cutting edges belong to $S$ surface, the primary peripheral surface of the end mill tool.

We have to notice that in the $I I$ motion, the $S$ surface is self-generated. So, this motion does not affect the generating process. The helical surface
generating is defined only by the assembly of motions $\quad I$ and $I I$.


Fig. 3. Generating kinematics

Regarding the position of points $M$ and $N$ onto the axial generatrix of the hub, there are defined the angles $\beta_{1}$ and $\beta_{2}$, as angles between the normals to the $G$ profile and $X$ axis.

It is denoted with $\beta$ the angle between the normal to the axial profile of the hub and the $X$ axis, in the contact point of the end mill with the $G$ generatrix, the point $P$ (see figure 2).

## 3. COORDINATES <br> TRANSFORMATIONS. ANALYTIC MODEL OF THE GENERATING TRAJECTORIES FAMILY

It is defined the relative position of the $X_{I} Y_{l} Z_{l}$ reference system regarding the $X_{0} Y_{0} Z_{0}$ axis, with $X_{I}$ axis of end mill:

$$
\begin{equation*}
X_{0}=\omega_{2}(\beta) \cdot X_{1} \tag{1}
\end{equation*}
$$

where $\beta$ is:

$$
\omega_{2}(\beta)=\left(\begin{array}{ccc}
\cos \beta & 0 & -\sin \beta  \tag{2}\\
0 & 1 & 0 \\
\sin \beta & 0 & \cos \beta
\end{array}\right)
$$

It is also defined the relative position of the $X_{0}^{\prime} Y_{0}^{\prime} Z_{0}^{\prime}$ and $X_{0} Y_{0} Z_{0}$ reference systems,

$$
\begin{equation*}
X_{0}^{\prime}=X_{0}-A \tag{3}
\end{equation*}
$$

$$
A=\left(\begin{array}{c}
-R+R_{0} \cdot\left(\cos \beta-\cos \beta_{1}\right)  \tag{4}\\
0 \\
-p \cdot \theta
\end{array}\right)
$$

The link between the $\beta$ angular parameter and the $\theta$ parameter is defined as:

$$
\begin{equation*}
\sin \beta=\frac{1}{R_{0}}\left(R_{0} \cdot \sin \beta_{1}-p \cdot \theta\right) \tag{5}
\end{equation*}
$$

The rotation motion around the $\vec{V}$ axis, with $\theta$ angular parameter, is given by:

$$
\begin{equation*}
X=\omega_{3}^{T}(\theta) \cdot X_{0} \tag{6}
\end{equation*}
$$

or, from (3),

$$
\begin{equation*}
X=\omega_{3}^{T}(\theta) \cdot\left[X_{0}-A\right] \tag{7}
\end{equation*}
$$

and, if consider equation (1), we obtain:

$$
\begin{equation*}
X=\omega_{3}^{T}(\theta) \cdot\left[\omega_{2}(\beta) \cdot X_{1}-A\right] \tag{8}
\end{equation*}
$$

representing the movement of $X_{l} Y_{l} Z_{l}$ space, joined with the end mill tool, regarding the $X Y Z$ space, joined with helix.

Developing we obtain the form:
$\left(\begin{array}{l}X \\ Y \\ Z\end{array}\right)=\left(\begin{array}{ccc}\cos \theta & -\sin \theta & 0 \\ \sin \theta & \cos \theta & 0 \\ 0 & 0 & 1\end{array}\right)\left[\left(\begin{array}{ccc}\cos \beta & 0 & -\sin \beta \\ 0 & 1 & 0 \\ \sin \beta & 0 & \cos \varphi\end{array}\right)\right.$.
$\left.\cdot\left(\begin{array}{c}X_{1} \\ Y_{1} \\ Z_{1}\end{array}\right)-\left(\begin{array}{c}-R+R_{0}\left(\cos \beta-\cos \beta_{1}\right) \\ 0 \\ -p \cdot \theta\end{array}\right)\right]$

After developments, we reached the form, see (5):

$$
\begin{aligned}
& X=\left[X_{1} \cos \beta-Z_{1} \sin \beta+R-\right. \\
& \left.-R_{0}\left(\cos \beta-\cos \beta_{1}\right)\right] \cos \theta-Y_{1} \sin \theta \\
& Y=\left[X_{1} \cos \beta-Z_{1} \sin \beta+R-\right. \\
& \left.-R_{0}\left(\cos \beta-\cos \beta_{1}\right)\right] \sin \theta+Y_{1} \cos \theta \\
& Z=X_{1} \sin \beta+Z_{1} \cos \beta+R_{0} \cdot\left(\sin \beta_{1}-\sin \beta\right)
\end{aligned}
$$

The equations (10) represent the generating trajectories family of points belonging to space $X_{I} Y_{I} Z_{l}$, joined with pre-formed tool, regarding the global reference system, joined with the helix.

If, from all the points belonging to the $X_{I} Y_{I} Z_{I}$ space, are selected those belonging to the peripheral primary surface geometric locus, so the relations (10) represent the generating trajectories family of $S$ surfaces regarding the helix - the generated blade surface.

### 3.1. Pre-formed tool surface - $S$

In figure 4 it is presented the axial section of pre-formed tool and the associated reference system.


Fig. 4. Axial tool's profile $-S$
The axial profile of the pre-formed tool is composed from two filled elementary profiles:
$-\overparen{A B}$, arc with equations:

$$
\begin{align*}
& \overparen{A B}: \left\lvert\, \begin{array}{l}
X_{1}=r_{0}-r_{0} \cos v ; \\
Y_{1}=0 ; \\
Z_{1}=-r_{0} \sin v,
\end{array}\right.  \tag{11}\\
& v_{\text {min }}=0 ; v_{\text {max }}=k . \tag{12}
\end{align*}
$$

Coordinates of $B$ point:

$$
\begin{align*}
& X_{1 B}=r_{0}-r_{0} \cos \kappa \\
& Y_{1 B}=0  \tag{13}\\
& Z_{1 B}=-r_{0} \sin \kappa .
\end{align*}
$$

$-\overline{B C}$, straight line profile, with equations:

$$
\overline{B C}: \left\lvert\, \begin{align*}
& X_{1}=r_{0}(1-\cos \kappa)+u \cdot \sin \kappa  \tag{14}\\
& Y_{1}=0 \\
& Z_{1}=-r_{0} \sin \kappa-u \cos \kappa .
\end{align*}\right.
$$

Revolving the equation assembly (11) and (14) around the $Z_{l}$ axis, the parametrical equations of the tool primary peripheral surface are obtained.

For $B C$ zone, the equations are obtained:

$$
S: \left\lvert\, \begin{align*}
& X_{1}=r_{0}(1-\cos \kappa)+u \sin \kappa  \tag{15}\\
& Y_{1}=\left(r_{0} \sin \kappa+u \cos \kappa\right) \sin \gamma \\
& Z_{1}=\left(-r_{0} \sin \kappa-u \cos \kappa\right) \cos \gamma
\end{align*}\right.
$$

with $\gamma$ angular parameter, at rotation around the $Z_{1}$ axis (the conic pre-formed tool's axis).

The equations assembly (10) and (15) represent the generating trajectory family for points belonging to $S$ tool surface, regarding the $X Y Z$ reference system, joined with helix.

### 3.2. Enwrapping condition for trajectories family

The enwrapping of the trajectories family (10), (15) represents the generated blade surface, meaning the blade of the rotor.

The variables, in the equations (10) and (15) assembly are: $\theta$-angular parameter for rotation around $Z$ axis; $\gamma$ - angular parameter for rotation around $Z_{1}$ axis; $u$ - linear parameter for translation along the cone generatrix.

In this case, the enwrapping condition for generating trajectories family (10) and (15) is expressed in form:

$$
\begin{equation*}
\left(\vec{R}_{\theta}, \vec{R}_{u}, \vec{R}_{\gamma}\right)=0 \tag{16}
\end{equation*}
$$

where vectors $\vec{R}_{\theta}, \vec{R}_{u}, \vec{R}_{\gamma}$ are partial derivative from (10), (15):

$$
\begin{align*}
& \vec{R}_{\theta}=\dot{X}_{\theta} \vec{i}+\dot{Y}_{\theta} \vec{j}+\dot{Z}_{\theta} \vec{k} ; \\
& \vec{R}_{u}=\dot{X}_{u} \vec{i}+\dot{Y}_{u} \vec{j}+\dot{Z}_{u} \vec{k}  \tag{17}\\
& \vec{R}_{\gamma}=\dot{X}_{\gamma} \vec{i}+\dot{Y}_{\gamma} \vec{j}+\dot{Z}_{\gamma} \vec{k} .
\end{align*}
$$

From (10) there are determined the partial derivatives:

$$
\begin{align*}
& \dot{X}_{\theta}=\left(-X_{1} \sin \beta \frac{d \beta}{d \theta}-Z_{1} \cos \beta \frac{d \beta}{d \theta}+\right. \\
& \left.+R_{0} \sin \beta \frac{d \beta}{d \theta}\right) \cos \theta-\left[X_{1} \cos \beta-Z_{1} \sin \beta+\right. \\
& \left.+R-R_{0}\left(\cos \beta-\cos \beta_{1}\right)\right] \sin \theta-Y_{1} \cos \theta \\
& \dot{Y}_{\theta}=\left(X_{1} \sin \beta \frac{d \beta}{d \theta}-Z_{1} \cos \beta \frac{d \beta}{d \theta}+\right.  \tag{18}\\
& \left.+R \sin \beta \frac{d \beta}{d \theta}\right) \sin \theta+\left[X_{1} \cos \beta-Z_{1} \sin \beta+\right. \\
& \left.+R-R_{0}\left(\cos \beta-\cos \beta_{1}\right)\right] \cos \theta-Y_{1} \sin \theta ; \\
& \dot{Z}_{\theta}=X_{1} \cos \beta \frac{d \beta}{d \theta}-Z_{1} \sin \beta \frac{d \beta}{d \theta}+R_{0} \cos \beta \frac{d \beta}{d \theta} .
\end{align*}
$$

It can be observed that, from (5), results:

$$
\begin{equation*}
\beta=\arcsin \left(\sin \beta_{1}-\frac{p}{R_{0}} \theta\right) \tag{19}
\end{equation*}
$$

and, so,

$$
\begin{equation*}
\frac{d \beta}{d \theta}=\frac{\frac{-p}{R_{0}}}{\sqrt{1-\left(\sin \beta_{1}-\frac{p}{R_{0}} \theta\right)^{2}}} . \tag{20}
\end{equation*}
$$

As well, the partial derivatives are defined regarding the $\gamma$ variable, see (15):

$$
\begin{align*}
& \dot{X}_{\gamma}=\left(\dot{X}_{1, \gamma} \cos \beta-\dot{Z}_{1_{1},} \sin \beta\right) \cos \theta-\dot{Y}_{1, \gamma} \sin \theta \\
& \dot{Y}_{\gamma}=\left(\dot{X}_{1_{y},} \cos \beta-\dot{Z}_{1_{\gamma}} \sin \beta\right) \sin \theta+\dot{Y}_{1_{\gamma}} \cos \theta  \tag{21}\\
& \dot{Z}_{\gamma}=\dot{X}_{1_{\gamma}} \sin \beta+\dot{Z}_{1_{\gamma}} \cos \beta
\end{align*}
$$

where, see (15), the partial derivatives are defined:

$$
\begin{align*}
& \dot{X}_{1_{\gamma}}=0 \\
& \dot{Y}_{1_{\gamma}}=\left(r_{0} \sin \kappa+u \cos \kappa\right) \cos \gamma  \tag{22}\\
& \dot{Z}_{1_{\gamma},}=\left(r_{0} \sin \kappa+u \cos \kappa\right) \sin \gamma
\end{align*}
$$

There are defined the partial derivatives regarding $u$ parameter:

$$
\begin{gather*}
\dot{X}_{u}=\left(\dot{X}_{1_{u}} \cos \beta-\dot{Z}_{1_{u}} \sin \beta\right) \cos \theta-\dot{Y}_{1_{u}} \sin \theta \\
\dot{Y}_{u}=\left(\dot{X}_{1_{u}} \cos \beta-\dot{Z}_{1_{u}} \sin \beta\right) \sin \theta+\dot{Y}_{1_{u}} \cos \theta  \tag{23}\\
\dot{Z}_{1_{u}}=\dot{X}_{1_{u}} \sin \beta+\dot{Z}_{1_{u}} \cos \beta \\
\dot{X}_{1_{u}}=\sin \kappa \\
\dot{Y}_{1_{u}}=-\cos \kappa \sin \gamma  \tag{24}\\
\dot{Z}_{1_{u}}=-\cos \kappa \cos \gamma
\end{gather*}
$$

### 3.3. The characteristic curve

The characteristic curve represents the contact curve between the $S$ surface, (15), and the blade surface.

The characteristic in the generating process, in the $X Y Z$ reference system, is obtained by associating the trajectories family (10), the enwrapping condition. In principle, the trajectories family has form:

$$
\begin{align*}
& X=X(\theta, \gamma, u) \\
& Y=Y(\theta, \gamma, u)  \tag{25}\\
& Z=Z(\theta, \gamma, u)
\end{align*}
$$

The enwrapping condition is:

$$
\begin{equation*}
\left(\vec{R}_{\theta}, \vec{R}_{\gamma}, \vec{R}_{u}\right)=0 \tag{26}
\end{equation*}
$$

In addition, for the geometric locus of the contact points to be a curve, the $\theta$ parameter has to be constant:

$$
\begin{equation*}
\theta=\text { const. } \tag{27}
\end{equation*}
$$

The (25), (26) and (27) equations assembly represents, in the $X Y Z$ reference system, the characteristic curve, the geometric locus of contact points between the $S$ surface and $\Sigma$, surface of rotor's blade, joined with the hub, but not defined yet.

In this way, for $\theta=0$, the assembly of partial derivatives is determined from (18):
$\dot{X}_{\theta=0}=-X_{1} \sin \beta \frac{d \beta}{d \theta}-Z_{1} \cos \beta \frac{d \beta}{d \theta}+$
$+R_{0} \sin \beta \frac{d \beta}{d \theta}-Y_{1} ;$
$\dot{Y}_{\theta=0}=X_{1} \cos \beta-Z_{1} \sin \beta+$
$+R-R_{0}\left(\cos \beta-\cos \beta_{1}\right) ;$
$\dot{Z}_{\theta=0}=X_{1} \cos \beta \frac{d \beta}{d \theta}-Z_{1} \sin \beta \frac{d \beta}{d \theta}+R_{0} \cos \beta \frac{d \beta}{d \theta}$.
and $\left(\frac{d \beta}{d \theta}\right)_{\theta=0}=-\frac{p}{R_{0} \cos \beta_{1}}$.
Also, the partial derivatives regarding the $U$ linear parameter, for $\theta=0$, from (23):

$$
\begin{align*}
& \dot{X}_{u(\theta=0)}=\dot{X}_{1 u} \cos \beta_{1}-\dot{Z}_{1 u} \sin \beta_{1} ; \\
& \dot{Y}_{u(\theta=0)}=\dot{Y}_{1 u}  \tag{29}\\
& \dot{Z}_{u(\theta=0)}=\dot{X}_{1 u} \sin \beta_{1}+\dot{Z}_{1 u} \cos \beta_{1} .
\end{align*}
$$

Similarly, the partial derivatives are calculated regarding the $\gamma$ angular parameter, for $\theta=0$, from (21):

$$
\begin{align*}
& \dot{X}_{\gamma(\theta=0)}=\dot{X}_{1 \gamma} \cos \beta_{1}-\dot{Z}_{1 \gamma} \sin \beta_{1} \\
& \dot{Y}_{\gamma(\theta=0)}=\dot{Y}_{1 \gamma}  \tag{30}\\
& \dot{Z}_{\gamma(\theta=0)}=\dot{X}_{1 \gamma} \sin \beta_{1}+\dot{Z}_{1 \gamma} \cos \beta_{1}
\end{align*}
$$

Now it is possible to write the enwrapping condition (26) as:

$$
\left|\begin{array}{ccc}
0 & \sin \kappa & -Y_{1}-\left(-X_{1} \sin \beta_{1}-Z_{1} \cos \beta_{1}+R_{0} \sin \beta_{1}\right) \cdot \frac{p}{R_{0} \cos \beta_{1}}  \tag{31}\\
\left(r_{0} \sin \kappa+u \cos \kappa\right) \cos \gamma & -\cos \kappa \sin \gamma & X_{1} \cos \beta_{1}-Z_{1} \sin \beta_{1}+R \\
\left(r_{0} \sin \kappa+u \cos \kappa\right) \sin \gamma & -\cos \kappa \cos \gamma & p-\left(X_{1} \cos \beta_{1}-Z_{1} \sin \beta_{1}\right) \cdot \frac{p}{R_{0} \cos \beta_{1}}
\end{array}\right|=0
$$

The condition (31) can be rewrite in form:

$$
\left|\begin{array}{ccc}
0 & -\tan \kappa & \left(X_{1} \sin \beta_{1}+Z_{1} \cos \beta_{1}-R_{0} \sin \beta_{1}\right) \cdot \frac{p}{R_{0} \cos \beta_{1}}  \tag{32}\\
\cos \gamma & \sin \gamma & X_{1} \cos \beta_{1}-Z_{1} \sin \beta_{1}+R \\
\sin \gamma & \cos \gamma & p-\left(X_{1} \cos \beta_{1}-Z_{1} \sin \beta_{1}\right) \cdot \frac{p}{R_{0} \cos \beta_{1}}
\end{array}\right|=0 .
$$

Further, equation (32) and surface's family (10), with condition $\theta=0$, is rewritten as:

$$
\begin{align*}
& X=X_{1} \cos \beta-Z_{1} \sin \beta+ \\
& +\left[R-R_{0}\left(\cos \beta-\cos \beta_{1}\right)\right]  \tag{33}\\
& Y=Y_{1} \\
& Z=X_{1} \sin \beta+Z_{1} \cos \beta
\end{align*}
$$

representing the characteristic curve on the blade's flank.

In relations (32), (33), $X_{l}, Y_{l}$ and $Z_{l}$ have meanings give by relations (15).

Really, the (33) equations represent functions depending from parameters $\gamma$ and $u$ :

$$
\begin{align*}
& X=X(\gamma, u) \\
& Y=Y(\gamma, u)  \tag{34}\\
& Z=Z(\gamma, u)
\end{align*}
$$

The (32) condition represents, in principle, an algebraic link between variables $u$ and $\gamma$, in form

$$
\begin{equation*}
\gamma=\gamma(u) \tag{35}
\end{equation*}
$$

Similarly, equations (15) and (32)assembly represent a spatial curve in the $X_{I} Y_{l} Z_{l}$ reference system, meaning the characteristic curve, identically to those from the blade flank.

Moving the characteristic curve:

$$
\begin{align*}
& X_{1}=X_{1}(u) \\
& Y_{1}=Y_{1}(u)  \tag{36}\\
& Z_{1}=Z_{1}(u),
\end{align*}
$$

with $u$ variable parameter, with law (8),

$$
\begin{aligned}
& X=\omega_{3}^{T}(\theta) \cdot\left[\omega_{2}(\beta) \cdot X_{1}-A\right] \\
& \text { with } \omega_{3}^{T}(\theta)=\left(\begin{array}{ccc}
\cos \theta & -\sin \theta & 0 \\
\sin \theta & \cos \theta & 0 \\
0 & 0 & 1
\end{array}\right) \\
& \omega_{2}(\beta)=\left(\begin{array}{ccc}
\cos \beta & 0 & -\sin \beta \\
0 & 1 & 0 \\
\sin \beta & 0 & \cos \beta
\end{array}\right) \text { and } \\
& A=\left(\begin{array}{c}
R+R_{0}\left(\cos \beta-\cos \beta_{1}\right) \\
0 \\
p \theta
\end{array}\right)
\end{aligned}
$$

The equation of the blade surface, in $X Y Z$ reference system, has parametrical form:

$$
\Sigma \left\lvert\, \begin{align*}
& X=X(u, \theta)  \tag{38}\\
& Y=Y(u, \theta) \\
& Z=Z(u, \theta)
\end{align*}\right.
$$

The generated blade shape is highlighted in plane sections perpendicular to the hub axis, in form:

$$
\begin{gather*}
Z=H, H-\text { variable }  \tag{39}\\
Z(u, \theta)=H \tag{40}
\end{gather*}
$$

equivalent with a principle form

$$
\begin{equation*}
u=u(\theta) \tag{41}
\end{equation*}
$$

In this way, the curves representing the generated blade profiles are defined by equations (matrix of coordinates):

$$
\Sigma_{H}=\left(\begin{array}{ccc}
X_{1} & Y_{1} & H  \tag{42}\\
X_{2} & Y_{2} & H \\
\vdots & \vdots & \vdots \\
X_{n} & Y_{n} & H
\end{array}\right), H \text { variable. }
$$

## 4. NUMERICAL APPLICATION

It is presented a numerical application for a rotor with characteristics:

- top radius of the hub $R_{t}=30.111 \mathrm{~mm}$;
- bottom radius of the hub $R_{b}=100 \mathrm{~mm}$;
- radius of hub's axial profile, see figure $2, R_{0}=150$ mm;
- angle $\beta_{I}=60^{\circ}$;
- angle $\beta_{2}=15^{\circ}$;
- helical parameter $p=100 \mathrm{~mm}$.

In order to graphically solve the problem, five crossing planes were generated, $P_{1} \ldots P_{5}$. These are crossing planes of the hub, where were drawn the segments, $L_{1} \ldots L_{5}$, with length corresponding to the radius of intersection circle between the spherical surface and the plan (see figure 5). The angle between these segments and the $X$ axis corresponds with the value gives by the helical parameter and the crossing plane elevation, according to the equation:

$$
\begin{equation*}
\theta=\frac{180^{\circ} \cdot H}{p \cdot \pi}[\mathrm{deg}] . \tag{43}
\end{equation*}
$$

Between the ends of these segments is drawn a spline curve, $G$, which will represent the trajectory of the mill tool axis end in its relative motion regarding the conical surface.

It is generated a suit of planes, $\pi_{l} \ldots \pi_{5}$, each $\pi_{i}$ plane being defined by two lines. One of these lines is the $Z$ axis, the other is the $L_{i}$ line.

In each of the $\pi_{i}$ plane, the spline curve is projected, obtaining the $\Gamma$ in-plane curve, see figure 5.

The toll axis position, for each of the $M_{1} \ldots M_{5}$ points, is obtained as perpendicular to the projected curve and passing through the current point $M_{i}, i=$ $1 . .5$, see figure 4.

A new suite of planes is generated, $P^{\prime}{ }_{1} \ldots P^{\prime}{ }_{5}$, perpendicular to the $G$ spline, each $P^{\prime}{ }_{i}$ plane passing through one end of the tool axis. In each of these planes is drawn a segment $r_{v i}$, perpendicular to the $L_{i}$ corresponding segment and with length equals to the tool top radius.

It is drawn a spline curve which materializes the trajectory of the second end of the tool axis, the $G_{I}$ curve, see figure 5.

A new suite of five planes is generated, $P^{\prime \prime}{ }_{l} \ldots$ $P{ }^{\prime \prime}{ }_{5}$, perpendicular to the $G_{l}$ spline curve. Each $P^{\prime \prime}{ }_{i}$ plane passes through the point which corresponds to the second end of mill axis.

In each of these planes is drawn a segment $r_{b i}$, perpendicular to the $L_{i}$ segment and with length equals to the tool bottom radius.


Fig. 5. Construction of geometric features needed for graphical solving of problem
There are drawn two spline curves, $G^{\prime}$, passing through the ends of $r_{v i}$ segments, and $G^{\prime}{ }_{1}$ passing through the ends of $r_{b i}$ segments, $i=1 \ldots 5$.

A sweep surface is generated, with two guiding curves, namely both spline curves, $G_{l}$ and $G^{\prime}{ }_{l}$.


Fig. 6. Rotor blade surface
The sweep surface is intersected with planes $Z=H$.

In figure 7 and table 1 are presented crossing sections of the blade, in planes $Z=H$, with $H$ variable.

Table 1. Coordinates of points from the crossing sections of the blade [mm]

| $\mathrm{H}=0$ [mm] |  |  | $\mathrm{H}=22.770$ [mm] |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: |
| X | Y | Z | X | Y | Z |
| 102.778 | 4.732 | 0.000 | 118.530 | 10.447 | 22.770 |
| 102.062 | 4.969 | 0.000 | 106.282 | 15.794 | 22.770 |
| 101.344 | 5.202 | 0.000 | 93.442 | 19.493 | 22.770 |
| 100.626 | 5.430 | 0.000 | 80.209 | 21.341 | 22.770 |
| 99.905 | 5.654 | 0.000 | 66.848 | 21.217 | 22.770 |
| $\mathrm{H}=45.540$ [mm] |  |  | $\mathrm{H}=68.311$ [mm] |  |  |
| X | Y | Z | X | Y | Z |
| 117.719 | 26.675 | 45.540 | 77.830 | 56.005 | 68.311 |
| 99.687 | 33.572 | 45.540 | 63.194 | 52.941 | 68.311 |
| 80.566 | 36.157 | 45.540 | 49.518 | 46.880 | 68.311 |
| 61.396 | 33.971 | 45.540 | 37.177 | 38.415 | 68.311 |
| 43.303 | 27.252 | 45.540 | 26.238 | 28.192 | 68.311 |
| $\mathrm{H}=91.081$ [mm] |  |  |  |  |  |
| X | Y | Z |  |  |  |
| 45.254 | 69.512 | 91.081 |  |  |  |
| 43.129 | 67.651 | 91.081 |  |  |  |
| 41.063 | 65.724 | 91.081 |  |  |  |
| 39.057 | 63.735 | 91.081 |  |  |  |
| 37.108 | 61.690 | 91.081 |  |  |  |



Fig. 7. Crossing sections of the rotor blade; points onto the crossing sections

## 6. CONCLUSIONS

This paper approaches in analytical form the issue of generating a blade of a rotor, which admits as hub a revolution surface with circular generatrix. The blade is generated with an end mill tool with a composed primary peripheral surface: a conical and a spherical surface.

The used method is the generating trajectories method, determining the composed surfaces family, in the reference system joined with hub.

The model of propeller blade surface is graphically generated, following the generating motion along the hub circular generatrix.

The methodology applied for the process is rigorous and easy to apply.

The presented methodology is applicable for complex shapes of hubs, too.

## ACKNOWLEDGEMENTS

This work was supported by a grant of the Romanian National Authority for Scientific Research and Innovation, CNCS - UEFISCDI, project number PN-II-RU-TE-2014-4-0031.

## REFERENCES

[1] S. P. Radzevich, Kinematics Geometry of Surface Machining. London: CRC Press, 2008.
[2] F. L. Litvin, Theory of Gearing, NASA, Reference Publication 1212, 1992.
[3] L. C. Chuang and H. T. Young, Integrated Rough Machining Methodology for Centrifugal Impeller Manufacturing, Int. J. Adv. Manuf. Technol., vol. 34, no. 11, pp. 1062-1071, 2007.
[4] T. Hideaki, T. Kawkubo, M. Tsukamoto, and R. Numakura, Development of High Efficiency Centrifugal Compressor for turbo Chiller, I.H.I. Eng. Rev., vol. 42, no. 2, 2009.
[5] X. Wei, Q. Du, and J. Liu, Research on Tool-Path Generating Methods for NC Machining of Centrifugal Pumps Vanes, Appl. Mech. Mater., vol. 26-28, pp. 982-987, 2010.
[6] Y. R. Wu and W. H. Hsu, A general mathematical model for continuous generating machining of screw rotors with worm shaped tools, Appl. Math. Model., vol. 38, pp. 28-37, 2014.

